

Lecture (4c)

Internal Combustion Engine

ENGINE PERFORMANCE

Performance Factors Of ICEs

The performance of an engine is evaluated on the basis of the following :

- **Specific Fuel Consumption.**
- **Brake Mean Effective Pressure.**
- **Specific Power Output.**
- **Specific Weight.**
- **Exhaust Smoke and Other Emissions.**

ENGINE PERFORMANCE

For an engine with:

Diameter (bore) D m

stroke length L m

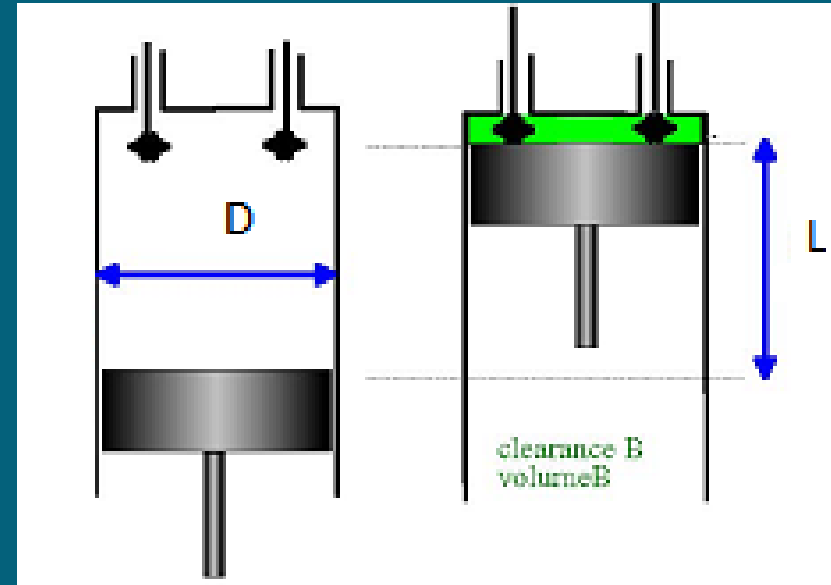
an engine speed N rpm

No. of cylinder n

No. of revolution for one cycle x

$x=2$ for 4-strok engine

$x=1$ for 2-strok engine



Air-Fuel Ratio:-

□ For combustion to take place, the proper ratio of air and fuel must be present in the cylinder.

□ The **air-fuel ratio** is defined as

$$AF = \frac{m_a}{m_f} = \frac{\dot{m}_a}{\dot{m}_f} \Rightarrow \frac{\text{kg}_a}{\text{kg}_f}$$

□ For a **SI** engine the AF is in the range of **12 to 18** depending on the operating conditions.

□ For a **CI** engine, where the mixture is highly non-homogeneous and the AF is in the range of **18 to 70**.

□ It is the ideal (Chemically Correct) A/F ratio that just burns all the fuel.

Natural gas: 17.2

Ethanol: 9

Diesel: 14.6

Gasoline: 14.7

Methanol: 6.4

Propane: 15.5

Hydrogen: 34

Equivalence Ratio:-

$$\phi = \frac{\left(\frac{F}{A}\right)_{actual}}{\left(\frac{F}{A}\right)_{chemically\ correct}}$$

$\phi = 1$ Chemically correct mixture

$\phi > 1$ Rich mixture

$\phi < 1$ Lean mixture

Calorific Value (CV):-

□ The amount of **heat liberated** by the combustion of a unit weight (or if a gas, a unit volume) of fuel.

Fuel	CV	Units
Gasoline	48000	kJ/kg
Diesel	45000	kJ/kg
Natural gas	43000	kJ/m ³

Fuel Power:-

□ Fuel power is the thermal power released by burning fuel inside the engine.

\dot{Q}_{Fuel} = Mass burned of fuel per second \times Calorific value of fuel

$$\dot{Q}_{Fuel} = \dot{m}_{Fuel} \times CV$$

$$\text{Units: } \frac{\text{kg}}{\text{s}} \times \frac{\text{kJ}}{\text{kg}} = \frac{\text{kJ}}{\text{s}} = \text{kW}$$

Power:-

The main purpose of running an engine is to obtain mechanical power.

➤ Brake power (b.p.):

The power developed by an engine and measured at the output shaft.

$$B.P. = \omega \times T = \frac{2\pi N}{60} \times T \Rightarrow \text{units: } \left(\frac{\text{rad}}{\text{rev}} \right) \left(\frac{\text{rev}}{\text{s}} \right) (J) = \text{Watt}$$

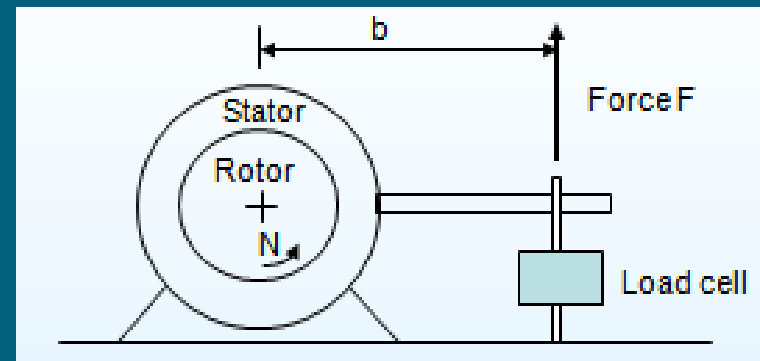
Where:

T	is torque	$N\cdot m$ or J
ω	is the shaft angular velocity	rad/s
N	is the rotational speed	revolutions per minute.

□ Torque is measured using a **dynamometer**.

The **torque** exerted by the engine is T :

$$T = F \cdot b \quad \text{units : } J$$

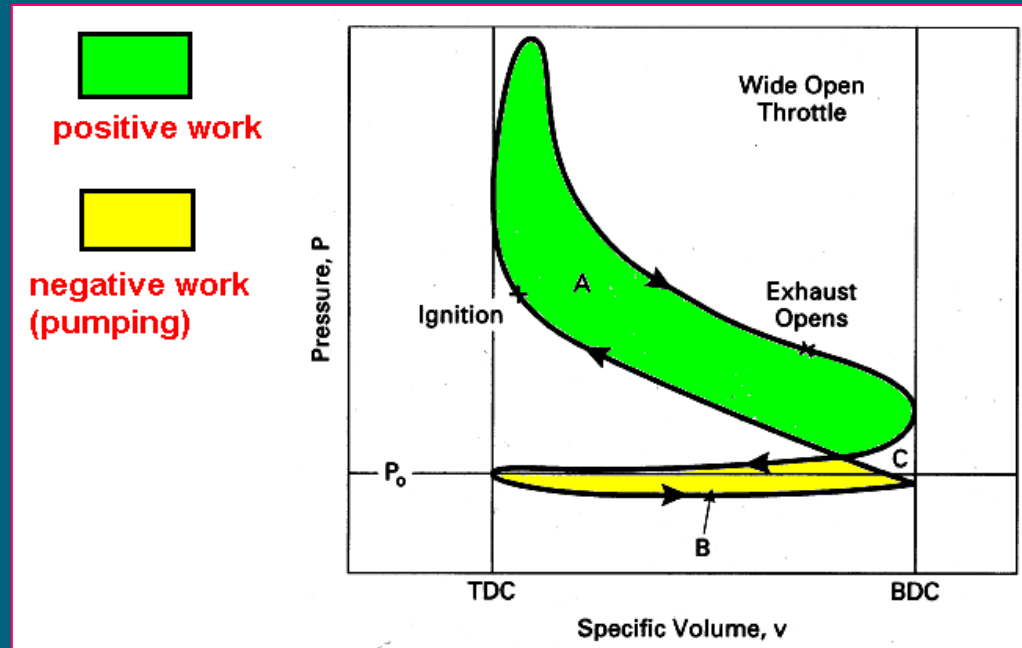


ENGINE PERFORMANCE

➤ Indicated power (i.p.):

It is the actual rate of work done by the working fluid on the piston.

$$I.P = \frac{W_i N}{60 \times x} \quad \frac{(kJ/cycle)(rev/s)}{rev/cycle}$$



➤ Friction power (f.p.):

The difference between the i.p. and the b.p and it is the power required to overcome the frictional resistance of the engine parts.

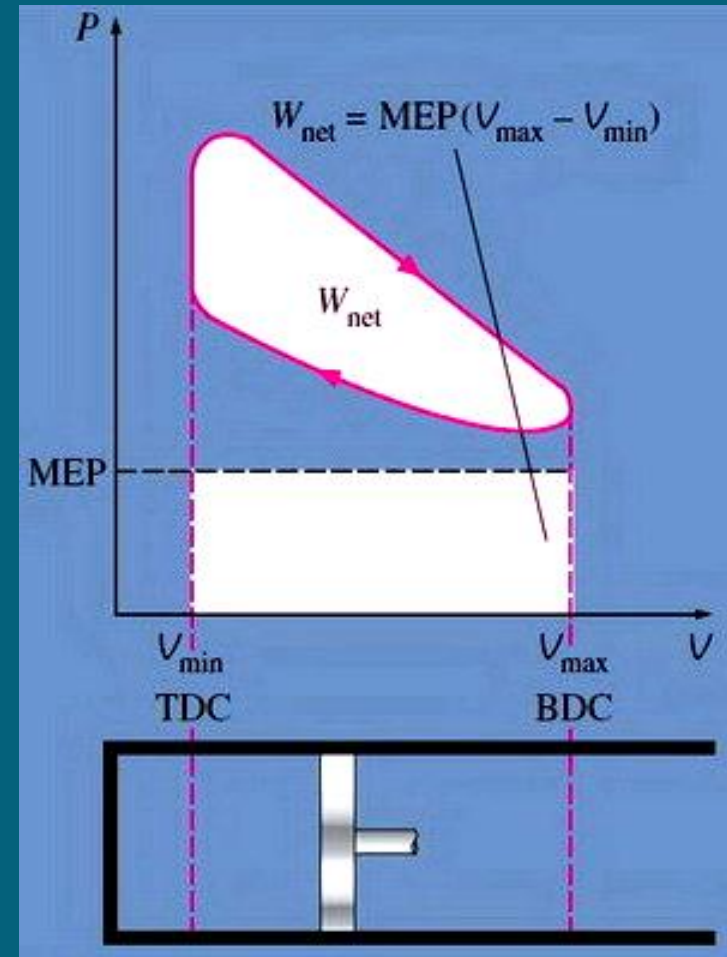
$$F.P. = I.P. - B.P.$$

mean effective pressure:-

Mean effective pressure is defined as a hypothetical/average pressure which is assumed to be acting on the piston throughout the power stroke.

$$mep = \frac{W_{net}}{V_d} = \frac{W_{net}}{V_{Total} - V_{Clearance}} \Rightarrow \text{Pa}$$

mep is a better parameter than torque to compare engines for design and output because it is independent of engine speed, N , and engine size, V_d .



➤ **Indicated mean effective pressure (imep):**

It is a hypothetical pressure which if acting on the engine piston during the working stroke would results in the indicated work of the engine.

$$\text{imep (P}_i\text{)} = \frac{\text{Net area of the indicator diagram}}{\text{Swept volume}} \times \text{Indicator scale}$$

or

$$\text{imep} = \frac{W_{\text{indicated}}}{\frac{\Pi}{4} D^2 L}$$

➤ **Brake mean effective pressure (bmep):**

It is the mean effective pressure acting on the pistons which would give the measured b.p.

$$\text{bmep} = \frac{W_{\text{break}}}{\frac{\Pi}{4} D^2 L}$$

➤ Fraction mean effective pressure (fmep):

$$fmep = imep - bmep$$

Note:-

- ❑ The maximum bmep is obtained at WOT at a particular engine speed.
- ❑ Closing the throttle decreases the bmep.
- ❑ For a given displacement, a higher maximum bmep means more torque.
- ❑ For a given torque, a higher maximum bmep means smaller engine.
- ❑ Higher maximum bmep means higher stresses and temperatures in the engine hence shorter engine life, or bulkier engine.
- ❑ For the same bmep 2-strokes have almost twice the power of 4-stroke

➤The power can also calculated by:-

Indicated Power

$$I.P. = imep \times \frac{\Pi}{4} D^2 L \times \frac{N}{60} \times \frac{n}{x}$$

Break Power

$$B.P. = bmep \times \frac{\Pi}{4} D^2 L \times \frac{N}{60} \times \frac{n}{x}$$

➤The mean effective pressure can also calculated by:-

Indicated mean effective pressure

$$imep = \frac{I.P.}{\frac{\Pi}{4} D^2 L \times \frac{N}{60} \times \frac{n}{x}}$$

Break mean effective pressure

$$bmep = \frac{B.P.}{\frac{\Pi}{4} D^2 L \times \frac{N}{60} \times \frac{n}{x}}$$

Specific fuel consumption (s.f.c.):

It is the mass of fuel consumed per unit power output per hour, and is a criterion of economic power production.

$$\text{sfc} = \frac{\dot{m}_f}{\text{Power}} \quad \text{kg/kW.hr}$$

Brake Specific fuel consumption

$$\text{bsfc} = \frac{\dot{m}_f}{\text{B.P.}}$$

Indicated Specific fuel consumption

$$\text{isfc} = \frac{\dot{m}_f}{\text{I.P.}}$$

Note:-

$$\text{bsfc} \cong 0.270 \quad (\text{Kg/kW.h}) \quad \text{for SIE}$$

$$\text{bsfc} \cong 0.200 \quad (\text{Kg/kW.h}) \quad \text{for CIE}$$

Specific Power Output:-

Specific power output of an engine is defined as the brake power (output) per unit of piston displacement.

$$\text{Specific Power Output} = \frac{B.P.}{V_d} = \frac{B.P.}{A_{piston} \times L_{strok}} = \frac{B.P.}{\frac{\pi}{4} D^2 L} \left[\frac{\text{kW}}{\text{m}^3} \right]$$

Volumetric efficiency (η_v):

the ratio of the volume of air induced, measured at the free air conditions, to the swept volume of the cylinder

$$\eta_v = \frac{m_a)_{act}}{m_a)_{the}} = \frac{m_a)_{act}}{\rho_a \times V_d} = \frac{m_a)_{act}}{\frac{P_1}{R.T_1} \times \frac{\pi}{4} D^2 L \times \frac{N}{60} \times \frac{n}{x}}$$

Thermal efficiency:-

$$\eta_{th} = \frac{Power}{m_f \times CV.}$$

Brake Thermal efficiency

$$\eta_{b.th} = \frac{B.P}{m_f \times CV.}$$

Indicated Thermal efficiency

$$\eta_{i.th} = \frac{I.P.}{m_f \times CV.}$$

Mechanical efficiency (η_m):

$$\eta_m = \frac{B.P.}{I.P.} = \frac{bmep}{imep} = \frac{isfc}{bsfc} = \frac{\eta_{b.th}}{\eta_{i.th}}$$